EVALUATION OF SINTERED METAL BRAKES FOR THE BAK-12 AIRCRAFT ARRESTING SYSTEM

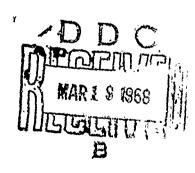
ALEX V. WOLFE, 1st LIEUTENANT, USAF

MAURICE E. PETERS

TECHNICAL REPORT SEC-TR-67-53

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DEPUTY FOR ENGINEERING
AERONAUTICAL SYSTEMS DIVISION
AIR FORCE SYSTEMS COMMAND
WRIGHT-PATTERSON AIR FORCE BASE, OHIO

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FOREWORD

This report was initiated by the Equipment Development Branch (SEMHD), Delivery and Retrieval Division, Directorate of Crew and AGE Subsystems Engineering of the Systems Engineering Group, Wright-Patterson Air Force Base, Ohio. This organization and the Air Force Flight Dynamics Laboratory (FDFM) monitored the test program. Lt. Alex V. Wolfe (SEMHD) was the officer in charge of the tests and Mr. Maurice E. Peters (FDFM) was the test engineer. The Deputy of Limited War (ASJT) of the Aeronautical Systems Division provided the test authority. The effort was conducted under Project 1559, "Limited/SAW Test and Evaluation."

The tests were conducted in the Landing Gear Test Facility of the Air Force Flight Dynamics Laboratory at Wright-Patterson AFB, Ohio, between 23 January 1967 and 6 March 1967. This report was submitted by the authors November 1967.

The E.W. Blisc Company loaned the sintered metal brakes to the Air Force for the tests and provided technical assistance, both at no cost to the Government.

This technical report has been reviewed and is approved.

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ABSTRACT

This report presents the results of a comparison test between a conventional B-52 disk brake and a newly developed sintered metal brake. The conventional B-52 brake, with cerametallic linings, is currently used on the B-52 aircraft and on the BAK-9 and BAK-12 aircraft arresting systems. The newly developed brake has sintered metal brake linings and is designed for the same applications as the conventional brake.

The objective of the test program was to compare the useful life and operating characteristics of the conventional brake with that of the sintered metal brake under conditions of simulated arrestments. The laboratory controlled conditions simulated a BAK-12 arrestment of a 35,000-pound aircraft at a speed of 150 knots,

The conventional brake yielded a useful life of 166 arrestments and the sintered metal brake had a life of 427 arrestments. The coefficients of friction for both brakes remained relatively constant throughout the test program.

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SECTION I

INTRODUCTION

A test program was initiated to compare the useful life and operational characteristics of a conventional B-52 disk brake with those of a newly developed, sintered metal brake when used on the BAK-12 arresting system. The conventional brake was made by the Bendix Corporation and the newly developed brake was made by the E.W. Bliss Company. Currently, the conventional brake is used on the B-52 aircraft and on the BAK-9 and BAK-12 arresting systems. The newly designed brake was developed for the same applications as the conventional brake. The general configuration for both brakes is shown in Figure 1. The BAK-12 portable arresting system is shown in Figure 2.

The Bendix disk brake utilizes cerametallic linings. It consists of four key-driven segmented steel rotors, three cerametallic stators, a cerametallic backing plate, a cerametallic pressure plate, and the piston or actuator housing. Figures 3 through 5 show the brake prior to use.

The E.W. Bliss Company has developed a sintered metal brake which consists of four key-driven solid rotors, three stators, a backing plate, and a pressure plate. See Figures 6 through 8. The wear pads (Figure 8) are riveted to a stator base plate, and the sintered brake lining material is bonded to the rotors. The Bliss brake can be assembled into the original B-52 housing without additional modifications.

The BAK-12 aircraft arrestor, manufactured by the E.W. Bliss Company, uses four B-52 brake assemblies in the two energy absorbing machines (Figure 1). One absorber is located on

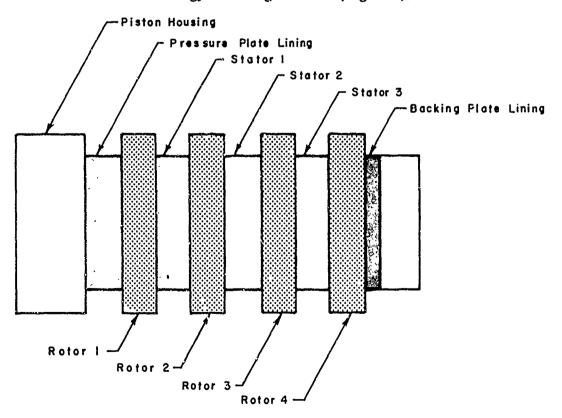


Figure 1. Brake Assemblies

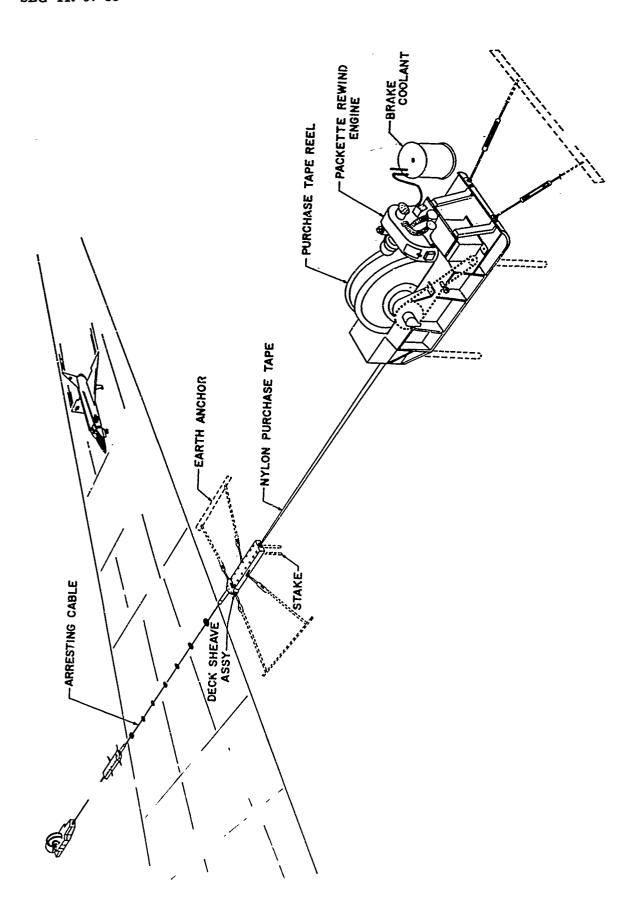


Figure 2. BAK-12 Portable Arresting System

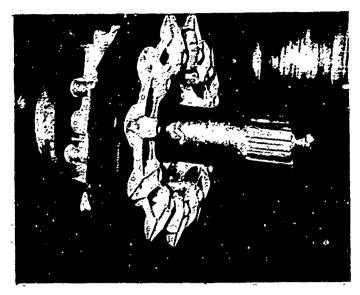


Figure 3. Bendix Brake Mounted on B-52 Test Axle

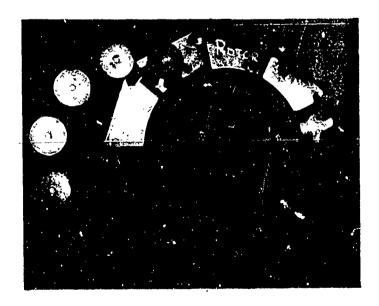


Figure 4. Bendix Rotor and Stator

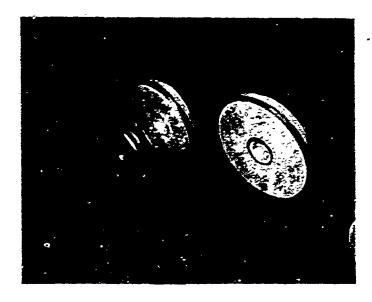


Figure 5. Cerametallic Lining Embodied in Metal Retaining Cup and Mounted on Stator

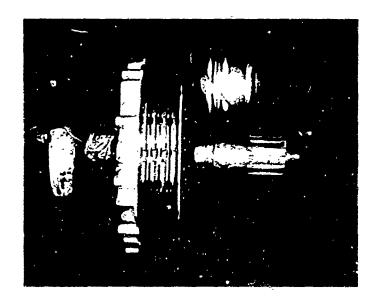


Figure 6. Bliss Brake Mounted on B-52 Test Axle

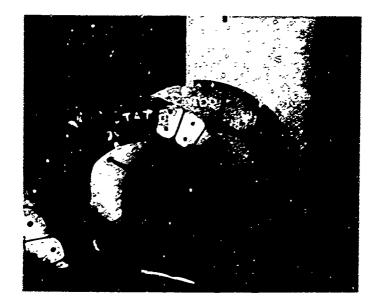


Figure 7. Bliss Rotor and Stator

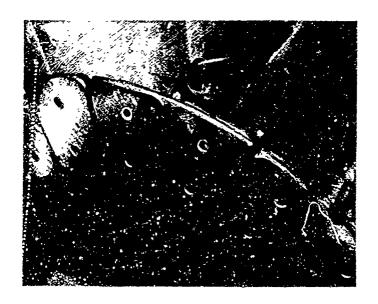


Figure 8. Bliss Riveted Wear Pads on Stator

each side of the runway and the absorbers are coupled together by a single steel cable and nylon purchase tapes. The purchase tape is wrapped on a storage reel which is splined to a shaft. Two B-52 brakes are mounted on a bearing block supporting the shaft, one on each side of the storage reel. As an aircraft engages the cable, nylon tape is pulled out and the brakes absorb the energy of the moving aircraft.

The 84-inch diameter inertia brake test dynamometer of the Landing Gear Test Facility in the Air Force Flight Dynamics Laboratory was used to conduct the tests.

SECTION II

TEST PROGRAM

The brake torque program was supplied by the E.W. Bliss Company. It consisted of two phases. The first phase was to conduct as many simulated arrestments on the Bendix brake as could be made. The test was to be terminated when the brakes were in a worn out condition or when erratic brake operation occurred. The second phase was to test the Bliss brake to the same conditions as the Bendix brake.

Simulated arrestments were conducted under laboratory-controlled conditions. These conditions were to simulate a 35,000-pound aircraft arrestment into the BAK-12 at 150 knots (34.8 \times 10⁶ ft-lbs energy). Since there are four brakes in the BAK-12 system and only one brake to be tested on the dynamometer, one-fourth of the total energy must be absorbed by one brake (8.7 \times 10⁶ ft-lbs).

SECTION III

DESCRIPTION AND OPERATION OF TEST EQUIPMENT

1. DESCRIPTION

The 84-inch diameter inertia brake dynamometer was selected for the test program because of its unique brake torque programming equipment and its adequate energy capacity.

The brake torque programmer is a closed servoloop operation consisting of a torque feedback signal, flywheel speed signal, brake pressure servovalve, and a function generator. The programmed torque (dependent variable) versus flywheel speed (independent variable) generates a command signal to the brake pressure servovalve. The servovalve then meters the pressure to the test brake as a function of the command signal to the torque feedback signal that the brake generates during an energy absorbing stop. The brake pressure is, therefore, a variable and is dependent upon speed and effective brake torque. The programmed speed versus torque is shown in Figure 9.

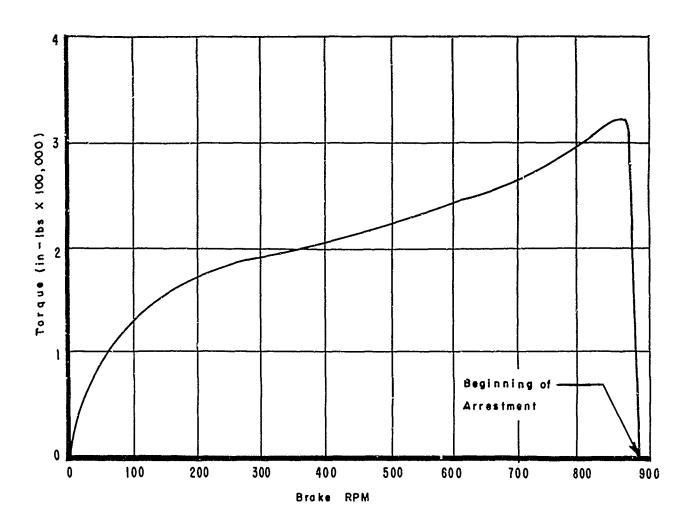


Figure 9. Programmed Torque Versus Brake RPM

A conventional B-52 wheel and tire had to be used to test the brakes on the inertia brake test dynamometer. Figure 10 shows a B-52 assembly loaded against the flywheel. The initial brake speed of the BAK-12 was met by the calculation of a rolling radius (RR) which was accomplished by calculating the ratio of the flywheel diameter to that of the RR of the tire and then equating the flywheel weight to the desired kinetic energy level. Calculations are shown in Appendix I.

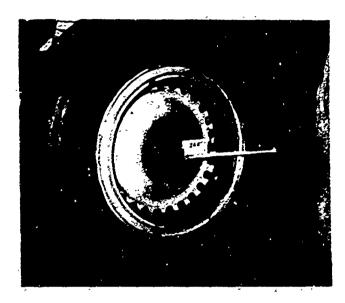


Figure 10. B-52 Wheel and Tire Assembly Landed Against Flywheel (Note RR 24.4 inches)

2. OPERATION

Once the dynamometer is programmed to the prescribed test conditions, the sequence of operations is semiautomatic. The operator presses a button which starts the operation. The flywheel is brought up to the prescribed landing speed of 512 RPM (equal to 882 brake RPM) at which time the wheel-brake and tire assembly is automatically landed against the rotating flywheel. When the prescribed load of 40,000 pounds and the RR of 24.4 inches are attained, the flywheel drive motor is automatically uncoupled from the DC drive source and the programmed brake torque automatically energizes the brake system. The braking system then brings the free spinning flywheel to a stop while following the prescribed program. During the braking cycle (simulated arrestment), the following parameters are recorded versus time: brake pressure, torque, flywheel speed, 'tire load and selected temperatures. An X-Y recorder records flywheel speed versus brake torque. Finally, a stop clock and flywheel revolution counter totalize the duration of the braking cycle.

SECTION IV

TEST RESULTS

1. BENDIX BRAKE 1

The first series of simulated arrestments, with the Bendix brake, were plagued with machine and operator difficulties; however, much was learned from this series of tests. In spite of the difficulties, the energy absorption per test run remained the same (8.7 x 10^6 ft-lbs). However, the rate at which the energy was absorbed deviated to a great extent from the programmed rate; therefore, a comparison with the Bliss brake performance was practically impossible. Higher torques were recorded during some of the tests due to excessive delay in the brake pressure application and air in the hydraulic system.

The brake was disassembled, inspected, and measured after 66 brake simulated arrestments. Figures 11 and 12 show the condition of the pressure plate lining which was the most severely worn lining, after 66 arrestments.

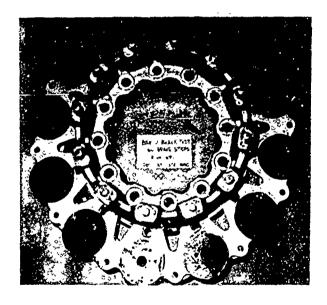


Figure 11. Pressure Plate Lining of Bendix Brake 1 After 66 Brake Arrestments

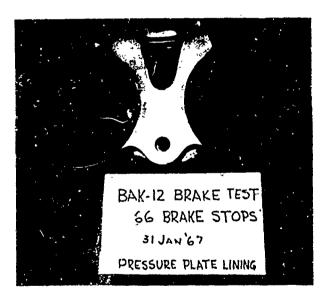


Figure 12. Most Heavily Damaged Brake Pad on the Pressure Plate

The preceding series of tests, conducted under laboratory controlled conditions, confirm the results obtained during field tests with actual aircraft during the first article BAK-12 evaluation. The report FTC-TDR-63-34 November 1963, Edwards AFB, page 34, clearly shows the same "breaking out" condition of the brake pads after 64 aircraft engagements.

The brake was reassembled and the tests were continued. A faulty thermocouple connection wrongly indicated that the brake was being adequately cooled in a 10-minute interval after each arrestment. Due to this faulty connection, arrestments 67 through 82 were conducted at intervals averaging 16 minutes. The 16-minute intervals varied from 5 minutes to 35 minutes. The time interval on the first 66 arrestments was 45 minutes. The brake performed satisfactorily during the short interval cycles.

After the error was discovered, the tests were continued through a total of 95 tests. Each test had a 45-minute cooling period. The tests were terminated at this point because of pulsating brake pressures.

The brake was disassembled and inspected again. Figures 13 and 14 show the condition of the pressure plate linings after 95 arrestments. It is interesting to note that the deterioration of the linings had not shown an appreciable increase from that of the linings which were taken after 66 arrestments (Figures 11 and 12) in spite of the high cycle rate of arrestments during Runs 67 through 82.



Figure 13. Pressure Plate Linings After 95 Simulated Arrestments



Figure 14. Most Heavily Damaged Lining on the Pressure Plate After 95 Arrestments (Same Lining as in Figure 11)

The brake pressures recorded throughout the 95 arrestments showed very little change in value or characteristic. A typical test as recorded by the X-Y recorder is shown in Figure 15. The brake pressure curve was superimposed on the curve after the arrestment.

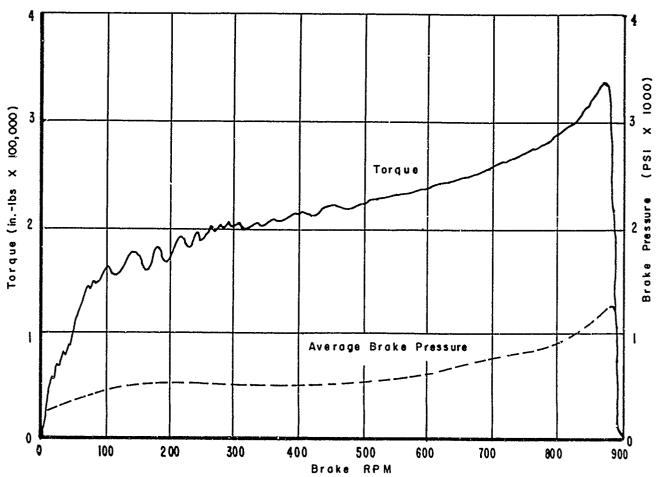


Figure 15. Typical Test Run on Bendix B-52 Brake

The results of the brake inspections after runs 66 and 95 are given in Tables I and II. These tables include the weights and measurements of the separate brake parts and also estimates of effective brake-lining area lost by damage to the cerametallic material during the arrestments. The estimates in Table I were made by several technicians working independently. There appears to be a discrepancy in the percent of area lost in the backing plate after 66 and 95 arrestments. This can be accounted for by noting that, when the material chipped out, the remaining material tended to "flow" into the vacant space due to the influence of high pressures and temperatures.

During the inspections after the 66th and 95th arrestments it was also noticed that many of the braking disks on the stators had become loose.

Figure 16 shows the results of a programmed constant torque test. This test was conducted to evaluate the brake effectiveness with heat buildup and has torque characteristics similar to those created during the braking of a B-52 aircraft. The brake pressure remained fairly constant throughout the test, indicating that the coefficient of friction of the lining remained constant throughout the heat buildup time. The constant torque test also provided a reference which was used to compare data received for a similar type arrestment for the Bliss brake.

TABLE I

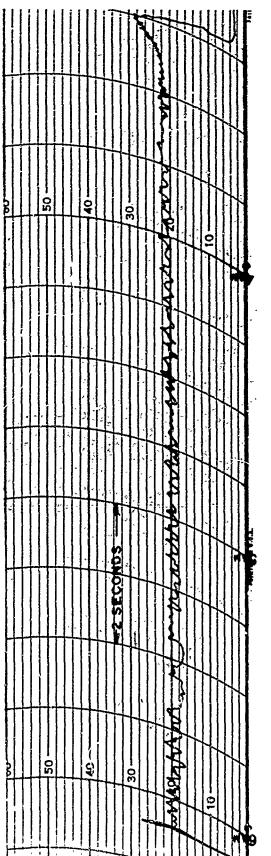
ESTIMATED PERCENT OF EFFECTIVE LINING AREA LOST DUE TO CHIPPED AND BROKEN OUT LINING MATERIAL

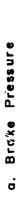
Lining	Percent after 66 arrestments	Percent after 95 arrestments
Pressure plate	15	18
Stator 1, rotor side 1	2	7
Stator 1, rotor side 2	1	3
Stator 2, rotor side 2	7	7
Stator 2, rotor side 3	1	2
Stator 3, rotor side 3	7	7
Stator 3, rotor side 4	1	1
Backing plate	5	4

TABLE II
WEIGHTS AND MEASUREMENTS FOR BENDIX BRAKE 1*

	We	Weights (pounds)	ds)		W	easureme	Measurements (inches)	(98)	
		After	After	New thickne	New thickness	Thickness after 66 tes	Thickness after 66 tests	Thickness after 95 tes	Thickness after 95 tests
	New	66 tests	95 tests	I.D.	0.0	I,D.	0.D.	I.D.	0.D.
Pressure plate lining	58 1/2	57 1/2	57 1/2	0.206	0.206	0.197	0.193	0.122	0.122
Backing plate lining	22 1/2	22	22	0.206	0.204	0.189	0,189	0.120	0.120
Stator 1	16	15	15	0.553	0.554	0.477	0.493	0.482	0.504
Stator 2	16	15	15	0.546	0.546	0.495	0.502	0.464	0.473
Stator 3	16	15	15	0.549	0.550	0.498	0.484	0.478	0.503
Rotor 1	24 1/2	24	24	0.367	0.370	0.363	0.366	0.358	0.368
Rotor 2	24 1/2	24	24	0.373	0.372	0.371	0.367	0.371	0.363
Rotor 3	24 1/2	24	24	0.371	0.372	0.369	0.368	0.368	0.367
Rotor 4	24 1/2	24	24	0.376	0.377	0,373	0.374	0.373	0.373

*See Figure 2 for schematic sketch of brake.





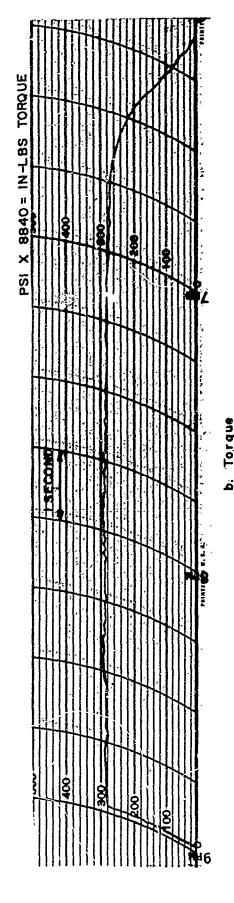


Figure 16. Constant Torque Data for Bendix Brake

The wear rate, which was determined from pin measurements at a brake pressure of 250 PSI, can be seen in Figure 17.

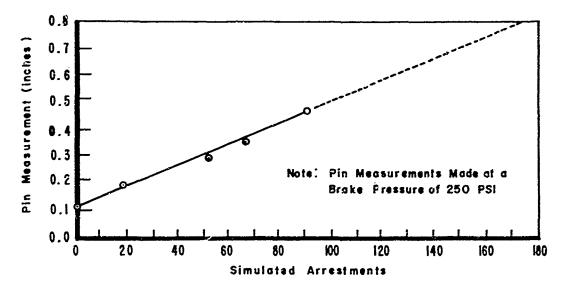


Figure 17. Wear Rate of Bendix Brake 1

2. BENDIX BRAKE 2

After termination of tests on the first Bendix brake, tests were begun on a second brake. The first 32 simulated arrestments were erratic, noisy, and excessive pulsations in the brake pressure recordings were indicated. The brake was disassembled and inspection revealed that the Number 3 rotor had a broken link and the Number 1 rotor was warped (see Figures 18 and 19). The wear pattern on Rotor 1 indicated that the segments were defective from the start of the test and were probably due to manufacturing deficiencies.

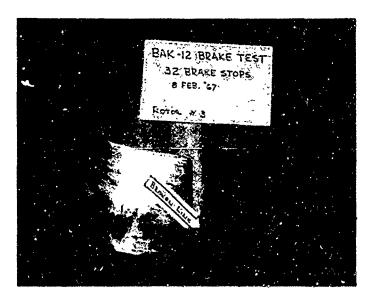


Figure 18. Rotor 3 and Missing Link

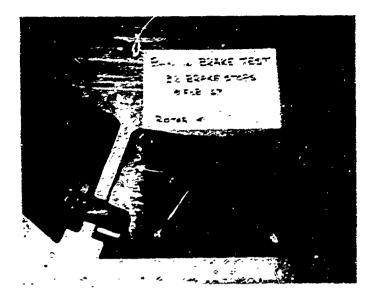


Figure 19. Wear Pattern of Rotor 1

The brake was reassembled, using Rotors 1 and 2 from the first Bendix brake to replace Rotors 1 and 3. Using worn rotors as replacement parts did not affect the brake operating characteristics because all the data duplicated the valid data from the first brake test.

This brake was not inspected at 66 stops because the information which could be obtained from such an inspection would duplicate the information obtained from the first brake. It was planned to disassemble the brake at 200 stops, but the program was terminated after 166 simulated arrestments. The test was terminated at this point because the specification for relining the brake limits the pressure plate travel to 23/32 of an inch (0.718 inch) and at 166 arrestments the measured travel pressure plate travel was 0.713 inch at a brake pressure of 250 PSI. The initial measurement was 0.109 inch. The average wear rate was 0.0036 inch per arrestment. See Figure 20 for a measure of wear rate throughout the brake life.

The brake was disassembled and weights and measurements were made (see Table III).

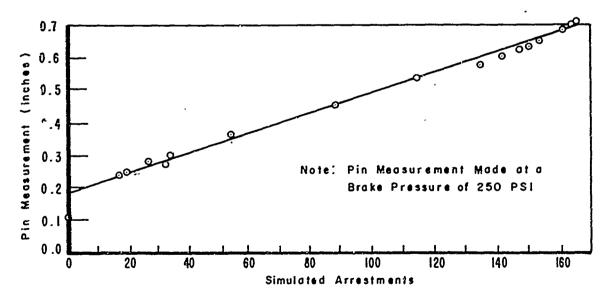


Figure 20. Wear Rate of Bendix Brake 2

TABLE III

WEIGHTS AND MEASUREMENTS FOR BENDIX BRAKE 2*

Lining	We	Weights (pounds)	(8)		~	Measurements (Inches)	nts (inchos	=	
		After	After	N thic	New thickness	Thickness after 32 tosts	nese 2 tosta	Thickness after 166 fee	Thicknose after 166 tests
	Ňew	32 tests	166 tests	I.D.	ο, υ,	T.D.	O,D.	I.D.	o.p.
Pressure plate lining	58-1/2	58	56	0.220	0.220	0.215	0.215	0.0()2	0.092
Backing plate lining	22-1/2	21	19	0.206	0.206	0.168	0,168	ł	1
Stator 1	16	16	13-1/2	0.553	0,554	0.518	0.528	0.399	0.465
Stator 2	16	16	13-3/4	0.546	0.548	0.528	0.538	0.425	0.441
Stator 3	16	16	14	0.549	0.550	0.51.9	0.525	0.430	0,432
Rotor 1	24-1/2	24-1/4	23-1/4	0.358	0.368	0.366**	0.370**	0,359	0,360
Rotor 2	24-1/2	24-1/4	23-1/4	0.373	0.372	0.364	0.368	0.357	0, 367
Rotor 3	24-1/2	24-1/4	23-1/4	0.371	0.363	0.371	0.371	0.365	0.361
Rotor 4	24-1/2	24	23-1/4	0.376	0.377	0.368**	0.371**	0.363	0.365

*See Figure 2 for schematic sketch of brake.

^{**}No. 1 rotor, segments warped, No. 3 rotor, broken link No. 1 and No. 3 rotors replaced with No. 1 and No. 2 Rotors from brake tests having 95 stops

Figures 21 through 24 show the condition of the linings at the conclusion of the test. The backing plate linings had the most amount of wear. Approximately 15% of the effective lining area was remaining (see Table IV).



Figure 21. Backing Plate Lining After 166 Arrestments

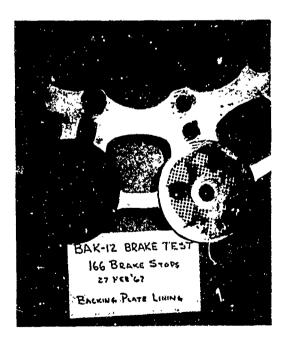


Figure 22. Closeup of Backing Plate Linings

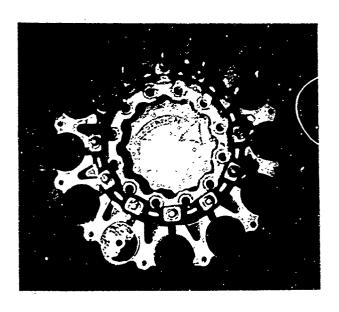


Figure 23. Pressure Plate Lining After 166 Arrestments



Figure 24. Closeup of Pressure Plate Lining

TABLE IV

ESTIMATED PERCENT OF EFFECTIVE LINING AREA LOST DUE TO CHIPPED OUT LINING

L	ining	Percent after 166 arrestments
Pressure	plate	17
Stator 1	Rotor side 1	20
Stator 1	Rotor side 2	40
Stator 2	Rotor side 2	12
Stator 2	Rotor side 3	25
Stator 3	Rotor side 3	20
Stator 3	Rotor side 4	12
Backing p	late	85

The coefficient of friction throughout the brake life was plotted in Figure 25. The coefficient was determined from the output torque and brake pressure at a point 2 seconds after the arrestment began (see Appendix I-2). From past experience, the coefficient of friction is known to decrease with brake life; however, it only decreased up to 30 arrestments and then remained at a constant value. It appears that the coefficient decreased up to the point where chipping out of the brake lining began. After chipping begins, a steel rim is exposed and serves as a steel-on-steel brake and this type of brake has a constant coefficient of friction, but a steel-on-steel braking surface is not an adequate brake.

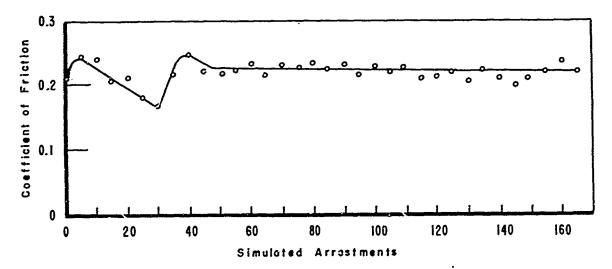


Figure 25. Coefficient of Friction for Bendix Brake 2

3. BLISS SINTERED METAL BRAKE

After completion of the Bendix brake tests, tests were begun on the E.W. Bliss Company sintered metal brake. The test program was identical to the one used for the Bendix brakes. After 62 and 200 simulated arrestments, the Bliss brake was disassembled for inspection of and comparison with the Bendix brakes.

The brake appeared to be in good condition after 62 simulated arrestments. The rotor linings had a good glaze and very few signs of chipping (see Figure 26). The wear pads on the stator showed signs of "dishing." "Dishing" can be defined as the turning up of the edges of the wear pads (see Figure 27). This effect is normal for wear pads that are retained as are the Bliss pads; that is, connected with two rivets at the center of the wear pad. The "dish" amounted to approximately 0.040 inch, which means that the wear pads will wear around their perimeter or outer edges (see Figures 27 and 28) more so than at the center.

During the first 62 simulated arrestments, temperatures were recorded from the four rotors through the use of thermocouples and slip rings. The peak temperature rise per arrestment was 400° ±10°F. This peak was reached within 30 seconds of the brake application. The thermocouples were placed in 1/16-inch diameter by 1-1/4-inch deep drilled holes in the outer edge of each rotor. The pressure plate and piston housing temperatures were also observed; the temperature rise on the pressure plate was 100° ±10°F and the rise on the housing was 10°F. An average of 45 minutes was required to cool the brake to 190° ±10°F which was the Bliss recommended maximum acceptable brake temperature prior to an arrestment. Seven thousand CFM of air at 70°F was directed across the brake well area as a means of cooling the brake assembly. The 45-minute cycle is required because of the brake confinement in the B-52 wheel.



Figure 26. Bliss Rotor 1 After 62 Arrestments

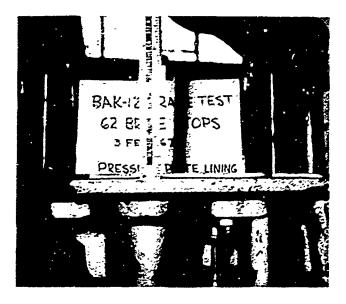


Figure 27. Dishing of Pressure Plate Wear Pad

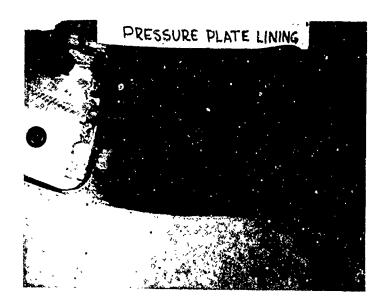


Figure 28. Wear Pattern of the Wear Pads on the Pressure Plate

A typical test, as recorded by the X-Y recorder, is shown in Figure 29. The brake pressure curve was superimposed after the test. A constant torque test was conducted to evaluate the brake effectiveness with heat build-up and has torque characteristics similar to those created during the braking of a B-52 aircraft. Figure 30 shows the results of this test. An analysis of the brake pressure curve indicates that the lining coefficient of friction increases with the imperature rise. This constant torque test was at the same energy level (8.7 x 10^6 ft-lbs) as the other programmed variable torque stops.

After 200 simulated arrestments the brake was still in good condition with the exception of Rotor 4. This rotor was "dished" approximately 0.025 inch across the wear surface (from the outer diameter to the inner diameter). The linings still had good glaze, but some chipping was beginning to occur. Figures 31 and 32 show Rotors 1 and 4 in the most damaged area. Very few changes had occurred since the inspection at 62 stops.

The Bliss brake tests continued until the brake would not perform properly. The brake torque showed a marked decrease during test 428 and was further verified during test 429. Therefore, a total of 427 successful simulated arrestments were made on the Bliss brake. The pin measurement was 0.676 inch at a brake pressure of 250 PSI.

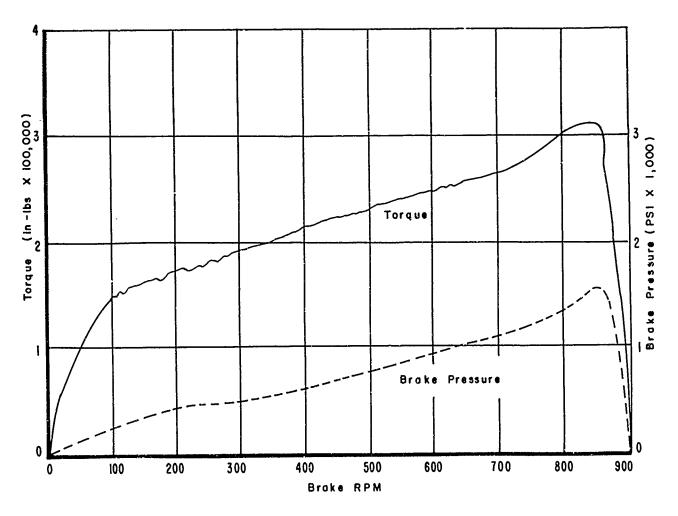
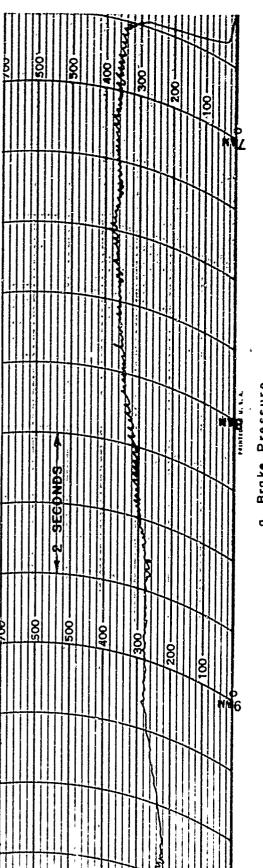
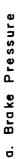
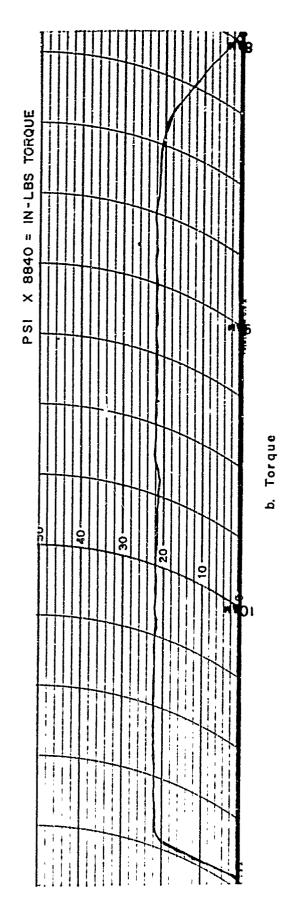


Figure 29. Typical Test Run on Bliss Brake







Constant Torque Data for Bliss Brake Figure 30.



Figure 31. Bliss Rotor 1 Showing Chipped Area After 200 Arrestments

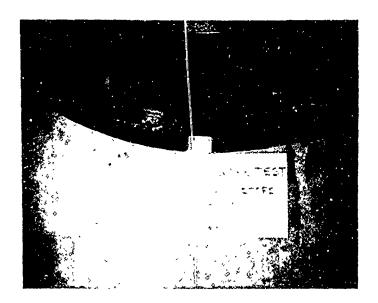


Figure 32. Bliss Rotor 4 Showing Chipped Area After 200 Arrestments

The brake was then disassembled for inspection and compared with the Bendix brake. Measurements as well as visual inspection revealed that the sintered material on the rotor was almost completely worn away (see Table V). There was much dust which had built up on the rotor, and, when it was cleaned off, it was evident that a few more successful arrestments could be made. However, the brake needed to be cleaned again within a few arrestments. Since this procedure was not practical, the test was not continued.

Chipping was very noticeable on the rotors, especially around the relief slots (see Figures 33 and 34).

The rotor had a pronounced glaze which indicates that the coefficient of friction remains constant. A flaking out of the wear pads caused a slight buildup of material on the rotor.

An investigation to discover any other brake deficiencies that might have occurred was conducted. The relief slots were examined for cracks and whether or not they had increased in width. Neither fault was found. Also the drive keys were studied to see if any noticeable battering or wear occurred. None was found.

Next the stators were examined. The most recognizable effect was the cracking of some of the wear pads (see Figure 35). This cracking was probably due to the "dishing" which causes the wear pads to wear mostly at the cutter edges. When these pads wear thin at the perimeter, they become structurally weak which in turn causes the pads to crack when brake pressure is applied.

"Dishing" of the wear pads was very pronounced. This effect resulted in the cracks on some of the wear pads as well as very little wear actually occurring at the center of the pads. The uneven wear and 'dishing" can be seen in Figures 35 and 36, respectively.

The brake pad material tended to "flow" over the pad edges. A combination of high temperatures and high pressures caused this flowing.

The backing plate withstood "dishing", wear, and flaking better than any other brake component. On the other hand the pressure plate exhibited the worst conditions of "dishing", wear, and flaking (see Figures 37 and 38).

The results of the measurements are presented in Table V. Several of the stators have an increased thickness. This increase is due to the "dishing" of the wear pads. However, there was a considerable amount of wear as indicated by the decreases in stator weights.

Figure 39 indicates the wear rate for the Bliss brake. An average wear rate of 0.00125 inch per arrestment was indicated for the Bliss brake.

The coefficient of friction remained relatively constant throughout the brake life (see Figure 40). The fluctuations at the beginning of the test program were caused by the "breaking in" of the brake. The characteristic of a constant coefficient of friction is highly desirable because the braking torque will not erratically change from arrestment to arrestment.

TABLE V

WEIGHTS AND MEASUREMENTS FOR SINTERED METAL BRAKE*

Lining		Weights (pounds)	s			*	Measurements (inches)	ts (inches	(1)		
		•		New	*	Thickness	ness	Thickness	ness	Thickness	ness
		V	V	thickness	ness	after 62 tests	tests	after 200 tests	0 tests	after 427 tests	7 tests
	New	200 tests	427 tests	l.D.	0.D.	l.D.	0.D.	1.D.	0.D.	I.D.	0.D.
Pressure plate lining	28 1/2	ŧ	27 3/4	0.412	0.412	1	0.420	ı	ı	0.114	0.114
Backing plate lining	58.0**	1	57.5	1	1	i	ı	ı	1	1	1
Stator 1	26 1/4	%	25	0.450	0.450	0.457	0.456	0.460	0.453	0.457	0.464
State: 2	26 1/4	79	25 1/2	0.451	0.451	0.457	0.456	0.465	0.457	0.442	0.439
Stater 3	26 1/2	Ж	25 1/2	0.451	0.451	0.457	0.458	0.459	0.459	0.464	0.466
Rotor 1	23	26 1/4	23 1/2	0.473	0.472	0.441	0.468	0.417	0.430	0.308	0.328
Rotor 2	8	26 1/4	23	0.474	0.472	0.446	0.455	0.407	0.430	0.341	0.315
Rotor 3	8	26 1/4	22 3/4	0.474	0.472	0.448	0.443	0.416	0.412	0.333	0.372
Rotor 4	33	28	22 1/2	0.471	0.469	0.445	0.440	0.427	0.390	0.334	0.318
Piston housing	90										
Total	313 1/2										ï

NOTE: Rotor thickness less lining = 0.313.

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*See Figure 2 for schematic sketch of brcke.
**Backing plate lining not measured because of housing configuration.

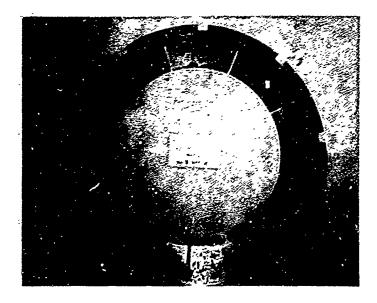


Figure 33. Bliss Rotor 3 Showing Chipped Area After 427 Arrestments

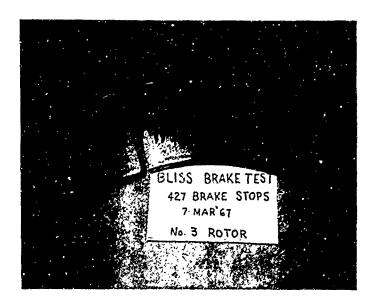


Figure 34. Closeup of Bliss Rotor 3 Showing Chipped Area After 427 Arrestments

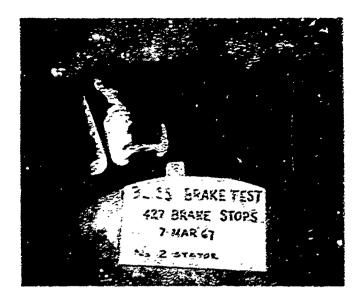


Figure 35. Bliss Rotor 2 Showing Cracks in Wear Pad After 427 Arrestments

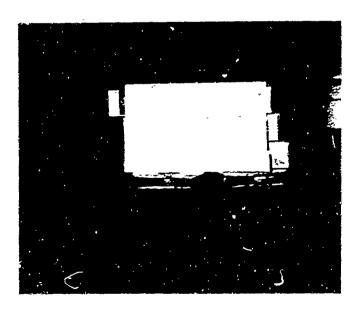


Figure 36. Dishing on Bliss Stator 1 After 427 Arrestments

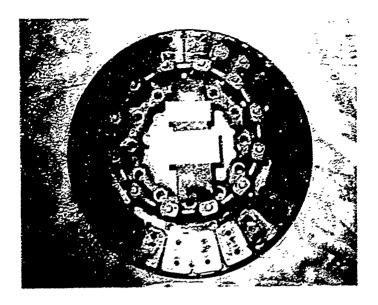


Figure 37. Bliss Pressure Plate After 427 Arrestments

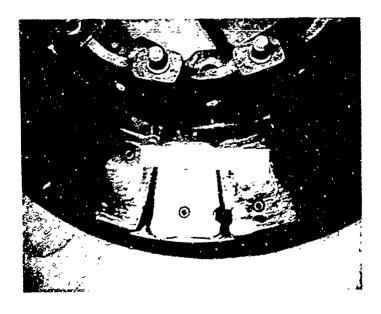


Figure 38. Closeup of Bliss Pressure Plate After 427 Arrestments

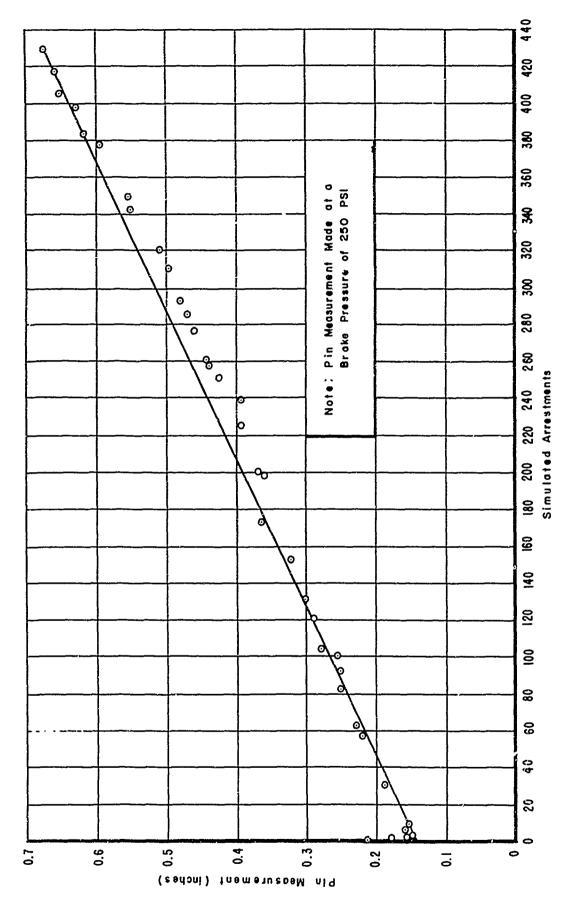


Figure 39. Wear Rate of Bliss Brake

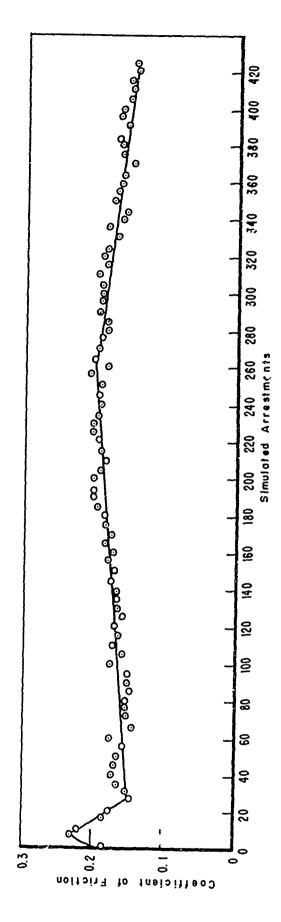


Figure 40. Coefficient of Friction for Sintered Metal Brake

SECTION V

CONCLUSIONS

The Bendix brake performed 166 simulated arrestments and the Bliss brake successfully completed 427 arrestments. The Bliss brake has a much smoother operation than the Bendix brake and is more suitable to the PAK-12 applications.

The "dishing" of the rotor and wear pads in the Bliss brake probably would not be as pronounced in actual application of the BAK-12 energy absorber, because brake pressure is maintained all the time so that tension can be maintained on the cable while the system is at standby.

A comparison of wear rates for the Bliss brake and the Bendix brake, illustrated in Figure 41, indicates that the sintered metal brake has 2 1/2 times as much life as the cerametallic brake.

The coefficient of friction for the Bendix brake decreases until the brake pucks begin chipping out and the Bliss brake coefficient remains relatively constant after the brake was broken in.

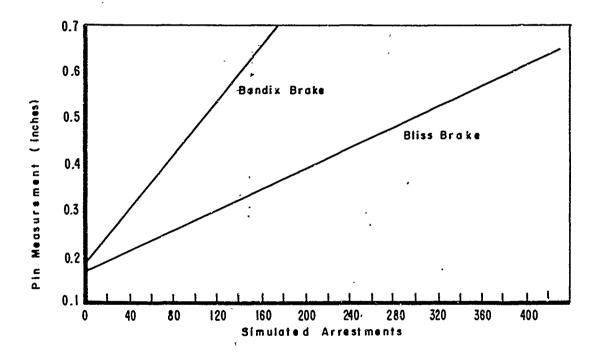


Figure 41. Wear Rate for Bendix and Bliss Brakes

SECTION VI

RECOMMENDATIONS

Based on efficiency, reliability, and life expectancy, the Bliss brake should be incorporated into the BAK-12 energy absorber.

The Bliss brake should be tested to the B-52 aircraft test conditions and evaluated for capability and reliability in this application. If the brake is tested in this application, the backing plate should be made of less weight and more flexible so that it can deflect with brake pressure. This should alleviate the tendency to "dish" and also decrease the uneven wear of the lining materials. However, the design of the backing plate must be carefully considered to insure a rigid construction. If the backing plate is allowed to bend under the normal load created by the brake pressure, the effective torque radius will change and affect brake performance. In turn the backing plate and adjacent rubbing surfaces will wear unevenly. This is evident from the severe damage to the backing plate of the Bendix brake.

The Bliss backing plate is of a much stiffer design to resist arresting gear application normal forces, which are much higher than B-52 application normal forces.

If it is decided to use the Bliss brake on the BAK-12, the brake linings should be replaced after a pin measurement 0.650 inch has been reached. This figure is somewhat lower than that which was actually measured in order to incorporate a factor of safety into the measurements.

APPENDIX I

CALCULATIONS FOR TEST PROGRAM

1. BRAKE TEST CALCULATIONS

Definitions

 $KE \stackrel{\triangle}{=} kinetic energy$

IE $\stackrel{\triangle}{=}$ inertia equivalent (weight at periphery of flywheel)

 $V \stackrel{\triangle}{=} \text{velocity (FPS)}$

 $RPM \stackrel{\Delta}{=} revolutions per minute$

RPS $\stackrel{\triangle}{=}$ revolutions per second

 $F \stackrel{\triangle}{=}$ force at periphery of flywheel

 $RR \stackrel{\triangle}{=}$ rolling radius (loaded tire measurement from center of axle to ground)

Desired Conditions

 $KE = 8.7 \times 10^6 \text{ ft-lbs}$

Initial brake RPM = 882 or 14.7 RPS

RR = 24.2 inches

Effective circumference = π x D = 3.14 x 2 x $\frac{24.2}{12}$

Effective circumference = 12.7 feet

Flywheel surface speed = circumference x RPS

Flywheel surface speed = 12.7 ft x 14.7 RPS

Flywheel surface speed = 187 FPS = 509 RPM

Calculations

$$KE = \frac{IE}{2g} \times v^2$$

IE =
$$\frac{64.4 \times KE}{V^2}$$
 = $\frac{64.4 \times 8.7 \times 10^6}{187^2}$

IE = 16,000 lbs

SEG-TR-67-53

The 84-inch flywheel, with 1 plates on the east side and 9 plates on the west side, has an IE of 15,901 pounds which is slightly less than the desired IE to compensate for this, the RR was changed. The new RR was determined by the following method:

$$V = \sqrt{\frac{64.4 \text{ KE}}{1E}}$$

$$V = \sqrt{\frac{64.4 \times 8.7 \times 10^6}{15,901}} = 187.7 \text{ FPS}$$

V = 512 RPM of flywheel

The RR must be corrected for the proper RPM.

Tire circumference =
$$\frac{\text{fl ywheel surface speed}}{\text{brake RPS}}$$

$$\pi \times 2 \times \text{RR} = \frac{187.7 \text{ FPS} \times 12 \text{ inch/ft}}{14.7 \text{ RPS}}$$

$$RR = 24.4 \text{ inches}$$

The inflation pressure of the tire was adjusted to a rolling radius of 24.4 inches at a tire load of 40,000 pounds.

2. COEFFICIENT OF FRICTION CALCULATION

Basic Equation

 $F = \mu N$

Definitions

 $F \stackrel{\triangle}{=}$ pulling force (pounds)

 $\mu \stackrel{\Delta}{=}$ coefficient of friction (dimensicnless)

 $N \stackrel{\triangle}{=} normal force on the brake (pounds)$

The pulling force (F) is determined by dividing the brake torque (T) by the distance from the brake center to the wear pads (R). The force (F) is then divided by eight because there are eight braking surfaces.

The normal force (N) is found by multiplying the brake pressure (P) by the total piston area (A).

Given Data for Run 50

Radius = 10.1 inches

Twelve pistons with a diameter = 1.375 inches

 $\mu = 0.167$

Brake pressure = 1.25×10^3 PSI

Brake torque = 3.0×10^5 inch-lbs

A =
$$12 \times \frac{\pi}{4} \times D^2 = 12 \times \frac{\pi}{4} \times (1.375)^2$$

A = 17.8 in^2
F = $\frac{3.0 \times 10^5 \text{ in-1bs}}{10.1 \text{ in} \times 8} = 3.71 \times 10^3 \text{ lbs}$
N = $1.25 \times 10^3 \text{ PSI} \times 17.8 = 2.22 \cdot 10^4 \text{ lbs}$
 $\mu = \frac{F}{N} = \frac{3.71 \times 10^3 \text{ lbs}}{2.22 \times 10^4 \text{ lbs}}$

APPENDIX II SUMMARY OF BRAKE TEST DATA

SUMMARY OF BRAKE TEST DATA

		7	T-				1			~		~					·					
Stop distance/	25 cycles (feet)		15.50	0 C	007:1	1184		1160	0017	1040	2 # OT . F.	1038	# 67 F	2113		\ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \	8027	1184	1170	1146	1137	1133
que,'	<u>L</u>		24.5	29.0	30.0	26.9		0 46	3 6	2. C	27.5	88.8	2 0	28. 28. 20.		04.0	2 6	0.7%	8.8	26.9	28.0	28.0
Braku torque,			4	, t.	2 00	, œ		œ	. 0	, G	0	G	α	; œ		c		x)	ω	ဖ	<u></u>	80
Braku 25 cycles	Max	3.1	48	46	. 4	31	CI	30		32	83 83	88	es:	. ee		23		2	31	32	32	83 63:
sure/ PSI)	Avg	llic Brake	1495	2460	1950	1131	lic Brake	1193	1148	1188	1145	1204	1178	1135	Brake	1134	1490	7 7 7	1434	1.389	1435	1434
Brake pressure, 25 cycles (PSI)	Min	ameta	100	175	125	1.10	ametal	100	100	75	0	85	90	100	Metal	125	100					35
Brak 25 c	Max	andix Cer	2500	2828	2725	1500	ndtx Cer	1825	1400	1400	1450	1450	1400	1400	Sintered	1625	1800	1750	0000	1690	1700	1725
Stop time/ 25 cycles	(seconds)	Data for Bendix Cerametallic Brake	11.0	10.9	10.8	11.1	Data for Bendix Cerametallic Brake 2	11.0	10.7	11.2	12.6	11.0	10.8	11.5	Data for Sintered Metal Brake	11.3	11,4		, ,	7.1.		11,4
Time between stops/25 cycles	(summed)		35	40	09	40		44	51	49	46	49	46	46		53	54	51	48	46	- -	40
Event or cvcle number			1~25	26-50	51-75	76-95		1-25	26-50	51-75	76-100	101-125	126-150	151–166		1-25	26-50	51-75	76-100	101-125	126-150	001-021

SUMMARY OF BRAKE TEST DATA (Continued)

Stop distance/	25 eyelsa (feet)		1100	1089	1003	1100	1066	1091	1098	1115	11:46	1153	1184	
ne/ x	Avg		28.3	28.6	28.3	27.3	27.9	28.7	28.4	27.8	31.0	29.3	27.0	
Brake torque/4.25 cycles (lbs x 10 ⁴)	Min		œ	œ	G	G	<u>r</u> -	c:	2	8	8	20	ဆ	
Bra 25 cyc]	Mux	(ned)	33	32	33	32	35	32	32	33	ဗ္ဗ	33	32	
ure/	Avg	e (Contin	1381	1321	1303	1241	1235	1249	1260	1295	1347	1399	1399	
Brake pressure/ 25 cycles (PSI)	Min	al Brak	75	09	75	75	75	75	75	75	75	75	75	
Brak 25 cg	Max	ered Met	1675	1575	1550	1500	1500	1560	1500	1550	1625	1700	1750	
Stop time/ 25 cycles	(seconds)	Data for Sintered Metal Brake (Continued)	11.4	11.6	11.4	11.2	11.2	11.3	11.4	11.3	11.5	11.5	11,2	
Time between stops/25 cycles	(minutes)		45	47	46	46	47	47	48	46	47	46	54	
Event or	cycle number		151-175	176-200	201-225	226-250	251-275	276-300	301-325	326-350	351-375	376-400	401-429	

APPENDIX III DATA FOR BENDIX CERAMETALLIC BRAKE 1

DATA FOR CERAMETALLIC BRAKE 1

	Romarks			No duta						Abort - system	trouble	Abort - system	trouble	Abort - system	tronoio		Abort - system		Abort - system trouble	Abort - system trouble	Abort - systom troublo
Flywhool	revolutions	1	7.9	1	533	56	57	70	57	53	111	09	115	32	100	7.1	42	78	92	င္ပ	29
10-4)	Min	22	21	1	1	10	12	12	14	0	17	0	30	0	ぜ	က	0	16	c	0	0
Torque (1bs x 10)	Max			-		30	30	30	30	12	ž	22	48	13	80	28	16	30	20	ဗ	ø.
ossuro I)	Min	0	100			175	160	250	250	0	460	0	0	0	175	195	0	255	0	0	c
Brake pressure	Max	200	750			1150	1100	1000	1775	1900	1575	2450	2050	2500	875	1950	2450	1250	2450	2450	2450
Stop time	(spuccos)	10.5	12.0	ı	11	11	13	11	10.0	0.3	14.0	7.0	11.6	0.5	16.0	14.2	1.0	11,0	2.5	0.5	a. a
Time between stops	(minutes)	t	1	1	ı	ı	55	30	20	40	98	40	40	40	15	40	40	65	40	99	10
Event	number	1	83	က	4	ល	9	7	8	6	10	11	12	13	14	15	16	17	18	19	20

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DATA FOR CERAMETALLIC BRAKE 1 (Continued)

	Потагкв	Pin moasured at 0, 190 inch at 250 psr.										Filter had to ba	מומוומים			Filtor had to be	paragra					Filtor had to bo oloaned
Flywhool	revolutions	58	09	79	53	23	67	00	57	09	മാ	89	20	98	131	102	28	59	59	62	62	62
Torque_4	Min	10	10	16	G	10	10	10	11	10	11	11	11	12	22	34	11	11	11	11	11	11
Tor (lbs x	Max	30	30	33	30	31	31	31	3.1	32	32	33	31	31	37	46	31	31	32	325	31	31
ressure SI)	Min	175	150	400	175	175	150	170	180	200	225	225	200	200	550	0	200	200	200	200	175	190
Brake pressure (PSI)	Max	2400	1150	1025	1425	1420	1120	1025	2350	2650	2625	2650	2700	1150	006	1550	2700	2750	2750	2325	2500	2700
Stop timo	(scconds)	11.1	11.2	9.1	11.0	11.1	11.2	11,2	10.4	10.9	10.3	10.8	10.5	12.4	14.0	13.0	10.4	10.7	10.6	10.6	10.9	11.0
Time between stops	(minutes)	50	180	35	25	30	40	30	40	40	40	40	40	40	40	40	40	40	40	20	40	40
Event	nuraber	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41

DATA FOR CERAMETALLIC BRAKE 1 (Continued)

	Romarks		Filter had to be			Filter had to be	7011010		Filter had to be	croming		Pin monsured at	0.287 inch. At 250 Par, boostor run charged and brake	Diga.							
Flywhool	novolutions:	69	7.1	57	63	29	28	63	92	69	ខ្លួ	29		58	និក	59	58	56	70	61	62
quo 10-4)	Min	10	11	11	10	น	10	10	11	11	11	11		11	10	11	ω	16	14	14	14
Torque (lbs x 10.	Max	32	33	31	32	32	31	32	32	31	32	32		30	30	30	30	30	29	30	30
ressure (I)	Min	200	175	175	175	300	175	195	200	195	195	195	 	200	170	160	125	300	275	300	300
Brake pressure (PSI)	Max	2650	2576	2775	2650	2650	2550	2650	2500	2825	2700	2575		1250	1260	1200	1260	1256	1150	1175	1250
Stop time	(seconds)	11.0	12.2	10.5	11.0	10.9	10.8	10,9	10.9	10.6	10.9	11.2		10.8	10.5	10.6	10.0	9.5	10.0	9.8	10.2
Time between stops	(minutes)	07	46	40	40	40	40	40	40	40	40	40		100	280	55	653	110	80	40	80
Event	number	42	43	44	45	46	47	48	49	20	51	52		53	54	55	56	57	58	59	09

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DATA FOR CERAMETALLIC BRAKE 1 (Continued,

	Romarks						Brake disassembled and inspected. Pin measured at 0.341 inch at 250 PSI.
Flywheel	revolutions	58	96	177	153	151	84
ne 4)	Min	16	14	38	38	59	13
Torque (lbs x 10 ⁻⁴)	Max	30	29	41	43	62	88
pressure (PSI)	Min	300	290	795			290
Brake pressure (PSI)	Max	1300	950	840			1050
	(seconds)	9.8	10,9	6.9	9.1	13.5	10.2
Time between stops	(minutes)	290	120	20	55	125	115
Event		61	62	ಕ್ಷಾ	64	65	· ` `

DATA FOR CERAMETALLIC BRAKE 1 (Continued)

-				
		SUR It	Max	255 255 255 255 255 255 255 255 255 255
		Prossure plate	Start	25555555555555555555555555555555555555
		on Ing	Мах	22222222222222222222222222222222222222
		Piston housing	Start	78888338888888888888888888888888888888
		5.4	Max	
	ires (•F)	Rotor	Start	Faulty theraccouple connection
	Temperatures (*F)	.3	Max	
	μ.	Rotor	Start	Faulty thermocouple connection ————
1		12	Мэх	25222222222222222222222222222222222222
		Rotor	Start	0.555.555.555.555.555.555.555.555.555.5
			Alax	\$5550000000000000000000000000000000000
		Rotor	Start	70 178 178 178 178 178 178 178 178 178 178
		i oduniti	revolutions	\$2 %25%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%
1		ue 10-4)	Min	<u> </u>
		Torque (1bs x 10-4)	Мах	######################################
	r.	sure	Min	125 125 125 125 125 125 125 125 125 125
	Rrake	pressure (PSI)	Мах	2725 2725 2400 2450 2450 2450 2450 1325 1325 1325 1325 1325 1325 1325 1325
		6	(seconds)	100.7 100.8 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0 100.0
		Time	minutes)	. 555999855956 - 52°59569956959
			number	\$ 250

*The temperatures from 67 through 82 are incorrect temperatures because the thermocouple and brake are making poor contact

** Pin measured at 0.460 inch at 250 PSI

*** Programmed for constant torque

* Test discontinued due to pulsating brake pressure

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APPENDIX IV DATA FOR BENDIX CERAMETALLIC BRAKE 2

DATA FOR CERAMITALLIC BRAKE 2

	ons Romarks	Pin measured at 0. 109 inch at 250 psr	4															Pin moasured at 0.230 inch at 250	4 1		Pin measured at 0.236 inch at 250
Flywheel	revolutions	52	51	54	54	55	20	54	52	53	20	51	53	51	20	52	52	52	57	ຄອ	33
Torque (lbs \times 10 ⁻⁴)	Min	10	10	<u></u> ∞	∞	∞	œ	&	80	∞	∞	ω	∞	80	ω	<u></u>	ω	∞	80	∞	ω
Tor (lbs x	Mex	30	30	30	30	30	30	30	30	30	30	30	30	30	30	30	30	30	30	30	30
ressure 31)	Min	200	200	140	125	125	200	100	100	100	100	100	100	110	100	100	100	100	100	100	100
Brake pressure (PSI)	Max	1825	1150	1175	1275	1150	1275	1300	1350	1300	1375	1400	1350	1410	1375	1375	1350	1390	1350	1400	1550
Stop time	(seconds)	11.0	11.0	11.1	11,1	11.1	11,1	11,1	11.0	11.1	11.0	11.2	11.0	11.2	10.9	11.1	11.1	11.4	11.0	11.2	11.0
Time between stops	(minutes)	t	40	40	40	40	40	40	40	40	40	40	40	40	40	40	40	40	40	40	40
Event	number	-	83	က	4	ಬ	9	2	8	6	10	11	12	13	14	15	16	17	18	19	20

DATA FOR CERAMETALLIC BRAKE 2 (Continued)

	Romarks							Pin measured at	PSI					Pin measured at 0.270 inch at 250 FSI. One link on #3 rotor failed. Brake was torn down and inspected. Motor #1 had indications of warped segments. Rotors 1 and 3 replaced with rotors 1 and 2 from Bendix brake No. 1 which had 95 arrestments. Pin measured at 0.294 inch at 250 PSI	
Flywheel	revolutions	54	53	55	54	56	52	99		62	57	57	53	50 48 49	
Torque_4	Min	8	6	G	10	o,	10	11		တ	G	o O	G	10 9 10	
Tor x sdl)	Max	30	30	30	30	30	31	33		30	30	30	30	32 32 33	7
ressure	Min	100	125	100	125	100	170	150		125	125	125	125	125 126 120	
Brake pressure (PSI)	Max	1425	1450	1450	1475	1350	1400	1300		1250	1325	1350	1400	1400 1275 1300	
Stop time	(seconds)	11.1	10.8	11.0	10.8	10.8	10.7	10.2		11.4	11.4	11.3	10.9	10.9	
Time between stops	(minutes)		30	55	09	09	20	80		20	40	55	ı	45 - 45	
Event	number	21	22	23	24	25	97	101		28	29	30	E & 3	32 33 34 33	

DATA FOR CERAMETALLIC BRAKE 2 (Continued)

	Remarks																				Pin measured at		
Flywheel	revolutions	48	48	48	49	48	48	47	48	47	49	47	48	48	49	48	49	47	50	50	49	49	45
1 8-	Min	10	10	10	10	10	10	10	10	10	10	10	10	10	10	11	6	6	11	11	11	11	10
Tore (lbs x	Max	32	33	33	32	32	32	32	32	31	31	31	31	31	31	31	31	31	31	31	32	31	31
essure I)	Min	100	100	100	100	100	100	100	100	100	100	100		100	100	100		06	100	110	110	100	06
Brake pressure (PSI)	Max	1300	1200	1175	1300	1350	1375	1350	1350	1350	1350	1325		1300	1300	1325		1325	1325	1350	1375	1375	1350
Stop time	(spuoses)	11.0	10.8	11.1	11.1	11.0	11.1	10.9	10.9	11.0	11.2	11,1	11.1	11.3	11.2	11,1	11.3	11.3	11.3	11.3	11.3	11.3	11,2
Time between stops	(minutes)	45	45	45	45	45	45	45	45	100	45	80	45	45	45	45	45	45	105	45	45	45	45
Event	number	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	20	51	52	53	54	55	56

DATA FOR CERAMETALLIC BRAKE 2 (Continued)

	Remarks																					Instrument air pres-	ממני פינים
Flywheel	revolutions	46	46	46	47	48	48	48	48	48	47	47	47	48	48	48	48	47	48	48	47	111	108
tue_4 10_4	Min	10	10	10	6	G	6		6	6	6	6	б	10	10	G	6	6	6	б	6	0	
ĭ× ï	ìi	32	32	32	31	31	31		32	32	32	32	32	32	32	32	32	32	32	32	32	13	
ressure II)	Min	06	06	100	06	75	06	75	75	75.	75	100	100	100	100	75	75	75	06	80	06	0	20
Brake pressure (FSI)	Max	1400	1400	1325	1350	1375	1375	1375	1400	1375	1400	1375	1375	1375	1375	1375	1350	1375	1350	1325	1350	775	775
Stop time	(seconds)	11.0	11.0	11.0	11.3	11.2	11.3	11.2	11.0	11.0	10.9	11.0	11.0	11.0	11.0	10.9	11.0	11.0	11.0	11.0	11.1	39.2	39.2
Time between stops	(minutes)	45	75	45	45	45	45	45	45	45	45	45	45	45	45	45	45	45	45	45	45	45	45
Event	number	57	58	59	09	61	62	63	64	65	99	29	89	69	20	7.1	72	73	74	75	92	7.2	& t~

DATA FOR CERAMETALLIC BRAKE 2 (Continued)

	Remarks	Brake pressure and torque were low. Filter was cleaned.								Pin measured at													
Flywheel	revolutions	77	50	48	48	48	20	48	49	48	48	47	47	47	47	47	48	48	48	48	48	46	
Torque 4	Min	∞	80	6	6	6	6	6	6	10	10	10	G	6	10	6	10	10	10	10	10	10	
Torc (lbs x	Max	13	31	32	31	32	32	32	32	32	32	32	33	32	32	32	33	33	32	32	32	32	
essure ()	Min	200	06	06	06	100	7.5	75	06	75	06	06	06	100	100	700	100	100	100	100	90	100	
Brake pressure	Max	775	1310	1360	1380	1400	1350	1350	1375	1400	1400	1450	1400	1400	1400	1400	1450	1400	1400	1400	1375	1400	
Stop time	(spuoses)	11.0	11.0	10.9	11.1	11.0	11.0	11.1	11.1	11.1	11.0	11.0	11.0	11.0	10.8	11.1	11.2	11.0	10.9	11.0	11.0	10.8	
Time	(minutes)	45	45	45	45	45	45	45	45	45	45	09	45	45	45	45	45	45	45	45	45	45	
Fvent	number	79	80	81	82	83	84	85	98	87	88	89	06	91	92	93	94	95	96	26	98	66	

DATA FOR CERAMETALLIC BRAKE 2 (Continued)

	Remarks						Brake pump was	torque or brake pressure, Attempt	was made twice.								Pin measured at	0.53.1 inch at 250 PSI.					
Flywheel	revolutions	50	48	49	48	48	48		48	49	48	49	48	51	49	49	47		49	48	48	48	49
)-41 ·	Min	6	10	တ	10	10	o		<u>ი</u>	10	6	6	၈	6	6	6	10		10	10	10	10	10
Tore (lbs x	Max	33	32	32	32	32	32		32	31	32	32	32	32	32	32	32	_	32	32	32	32	32
ressure I)	Min	100	100	100	100	100	90		06	06	06	06	06	85	06	06	100		100	100	100	100	100
Brake pressure (PSI)	Max	1425	1425	1350	1375	1425	1350		1350	1360	1380	1360	1410	1350	1390	1360	1375		1375	1425	1350	1325	1400
Stop time	(seconds)	10.8	10.8	11.0	11:0	11.0	11.1		10.9	11.0	10.9	11.0	10.9	11.0	11.0	11.1	11.0		11.0	11.1	11.0	11.1	10.8
Time between stops	(minutes)	45	45	45	45	45	06		45	ن و	45	45	45	45	45	45	ı		45	45	45	45	75
Event	numper	100	101	102	103	104	105		106	107	108	109	110	111	112	113	114		. 115	116	117	118	119

DATA FOR CERAMETALLIC BRAKE 2 (Continued)

	Remarks															Pin measured at								Pin measured at 0. 608 inch at 250 PSI.
Flywheel	revolutions	20	50	49	20	20	49	51	20	49	15.1	20	52	50	51	49	20	49	49	50	09:	20	20	51
que 10 ⁻⁴)	Min	10	10	10	6	10	6	6	6	6	တ်	6	6	6	6	6	6	6	6	10	10	10	10	10
Torque (lbs x 10	Max	32	32	32	32	32	32	32	32	32	32	32	32	32	31	32	31	32	31	32	32	32	32	32
ressure	Min	100	100	100	06	100	100	100	100	190	100	100	100	100	100	100	100	100	06	110	100	100	110	100
Brake pressure (PSI)	Max	1400	1450	1460	1400	1425	1325	1390	1400	1400	1375	1400	1400	1400	1340	1355	1300	1325	1325	1375	1375	1340	1375	1325
Stop time	(seconds)	11.0	11.0	11.0	11.1	10.9	11.0	11.0	11.1	11.2	11.2	11.2	11.3	11.1	11.1	11,1	11.4	11.2	11.1	11.0	11.3	11.2	11.3	11.2
Time between stops	(minutes)	45	45	45	45	65	45	45	45	45	45	45	45	45	45	45	45	45	45	45	45	45	45	45
Event	number	120	121	122	133	124	125	126	127	128	129	130	131	132	133	134	135	136	137	138	139	140	141	142

DATA FOR CERAMETALLIC BRAKE 2 (Continued)

-								
Time	Time between "tops	Stop time	Brake pressure (PSI)	ressure I)	Tor (lbs x	Torque $_{3}$ x $_{10}^{-4}$	Flywheel	
Ħ	(minutes)	(seconds)	Max	Min	Max	Min	revolutions	Remarks
45		11.3	1325	100	31	10	52	
45		11, 1	1325	110	33	11	51	
40		17.1	1250	100	32	10	49	
45		12.0	1350	125	32	12	28	
45		11.6	1375	125	32	12	57	Pin measured at
45		11,5	1325	100	31	∞	49	
45		11.4	1325	100	31	∞	49	
75		11.2	1375	100	31	6	48	
45		11.4	1356	100	31	6	49	Pin measured at
45		11.5	1400	100	32	6	50	
45		11.5	1325	100	31	∞	49	
45		11.5	1350	100	32	6	20	Pin measured at 0.649 inch at 250 PM.
55		11.2	1325	100	32	o.	49	
45		11.4	1250	100	31	6	50	
45		11.4	1200	125	31	∞	51	
45		11.4	1200	125	32	∞	51	
45		11.2	1240	140	33	ø	51	
45		11.0	1250	125	33	6	52	
45		12.0	1250	140			53	
55		11.2	1400	125	33	6	51	Pin measured at
40		12.0	1375	125	32	6	51	

DATA FOR CERAMETALLIC BRAKE 2 (Continued)

	Remarks		Pin measured at	0. 705 inch at 250 PSL. Pin measured at	0.713 inch. Test	terminated because	brake had worn out.	
Flywheel	revolutions	51	51	50				
Torque (bs \times 10 $^{-4}$)	Min	6	6	တ				
Tor x 3dl)	Max	32	32	32				
Brake pressure (PSI)	Min	125	105	125	-			
Brake pre	Max Min	1275	1350	1325				
Stop time	(seconds)	12.1	11.1	11.4				
Time between stops	(minutes)	50	45	45				
Event	number	164	165	166				

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APPENDIX V DATA FOR BLISS SINTERED METAL BRAKE

		1	
			Pin measured at 0, 216 litch at 25°, 7-30 Pin measured at 0, 179 litch at 250 Pin measured at 0, 179 litch at 250 Pin measured at 0, 150 litch at 250 Pin measured at 0, 160 litch at 250 Pin Pin measured at 0, 160 litch at 250 Pin measured at 0, 162 litch at 250 Pin measured at 0, 165 litch at 250 Pin measured at 0, 166 litch at 250 Pin measured at 0, 168 litch at 250 Pin measured at 0, 168 litch at 250 Pin Motor 4 thermocouple falled Pin measured at 0, 188 litch at 250 Pin Measured at 0, 188 litch at 250 Pin Hotor 4 thermocouple falled Removed filler; Rotor 2 thermocouple falled Removed filler; Rotor 2 thermocouple falled
	2 2	ž	25
	Pressure plate	Start	8
	c 3	Max	
	Piston housing	Start	\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$
tures	•	Max	8 4 4 4 5 5 5 5 8 8 8 8 8 8 8 8 8 8 8 8
Temperatures	Rotor	Start	Faulty Thermocouple Connection
F	Rotor 3	Start Max	Faulty Thermocouple Connection
	72	Max	5 1 1 5 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8
	Rotor	Start	Faulty Thermocouple Connection
	-	Max	5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5
	Rotor 1	Start	180 180 180 180 180 180 180 180 180 180
		revolutions	25555555555555555555555555555555555555
	(lbs x 10-4)	Mfn	7770 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
1	9 €9	Max	
4	essure (PSI)	Min	\$288
	pressure (PSI)	Max	1655 1000 1000 1000 1000 1000 1000 1000
	9 11	(spuccos)	
		(minutes)	1 % 1 % % % 4 % % 4 % % % % \$ 6 % 8 % 8 % 8 % 8 % 8 % 8 % 8 % 8 % 8 %
	į	number	

		5	COMMING	Pin measured at 9, 220 thoh at 250 PSI	Pin measured at 0, 230 inch at 250 PSI, Brake torn down and inspected.	Thermocouples removed. Constant torque stop	Pin messured at 0, 252 Inch at 250 PSI	Pin measured at 0, 252 inch at 250 PSI	Pin measured at 0, 259 inch at 250 PSI Wheel skidded - cause was unknown	Pin measured at 0.880 inch at 250 PSI
	2 0	یا	į	160 175 195 195 195	200					
	Pressure plate		Start	258885	85					
	E 3		X	88888	55					
	Platon		Li g	555558	88					
tures	-		ž							
Temperatures	Rotor		L MA			Therms couple Connection	wilus?			
٢	1	2	ž	·			form -			
	Robor	O P	E E			Thermocouple Connection	wilned			i
	1,		ğ							
	Botor	Won.	Start			Thermocouple Connection	Allued			
		1	Mex	575 595 565 555 575	540					
	٩	HOTOF 1	Start	55 55 55 55 55 55 55 55 55 55 55 55 55	110					
		Flywheel	revolutions	52 54 54 54 54	6 6	\$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$	288888888888	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	8 8 8 8 8 2 2 2 2 2	22232
	en K	ş .	МП	တ ၊ ၈၈ ၈ ၈	n co co	ပ်သတ္ထတ္ထတ္တ ၊ ထထလလလ	n) လာ	
	Torque (ibs x	٤	Max	ន្ទាននេះ	888			ន ន្តន្តន្តន្តន	*********	*****
	aure sure	Ę,	Min	011	333	11000000000000000000000000000000000000				
	Brake	ë	Max	1700 1650 1650 1675	1750	1650 1650 1650 1660 1660 1650 1650 1650	1650 1650 1650 1690 1610 1610 1610	1500 33 1600 1600 33 1600 1600 33 1600 1600 33	1650 1650 1650 1650 1575 1550 1650 1675	1650 1650 1673 1673
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		Romarks	Ph measured at 0, 290 inch at 250 PSI	Pin measured at 0, 300 inch at 250 PSI	Pin messured at 0,323 inch at 250 PSI
	Pressure plate	Start Max			
					
	Piston housing	Start Max			
	- 2				
Tomperatures	Rotor 4	rt Max	uple Connection	Faulty Thermoco	-
Ton	£	x Start		··	
	Potor 3	Start Max	uple Connection	Fauly Thermocon	
	Rotor 2	Start Max	uple Connection	Faulty Thermoco	
	Rotor 1	Start Max			
	Flourbee	revolutions	22 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8	2 4 8 4 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8
	(lbs x 10-4)	Min		ೲೲೲೲೲೲೲೲ	
	282	Max		<u> </u>	
	essure (PSI)	Mh			857777777777777777777777777777777777777
	Ĩ.	Max	1625 1650 1650 1650 1700 1650 1675 1675 1675 1675 1676 1676 1676 1676	1675 1875 1625 1625 1620 1620 1620 1720	1625 1650 1650 1650 1650 1650 1650 1650 165
		(seconds)			
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	į	number	109 1110 1111 1111 1116 1116 1118 1120 1120 1120 1120 1120 1120 1120	128 128 133 133 134 134 135 135	133 145 145 146 146 146 146 146 153 153 153 153 154 156 157 160 160

		Remarks	Pin measured at 0,363 inch at 250 PSI	Instrument air pressure was too low	Pin measured at 0,362 inch at 250 PSI Brake disassembled and inspected Pin measured at 0,370 inch at 250 PSI
	Pressure plate	Max			
	ard d	Start			
	Piston bousing	Start Max			
	Piston housing	Start			
Temperatures	Rotor 4	Max			
Tempe	Rot	Start			
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	1 1	XVIV			
	Rotor 1	Start			
	- Inches	revolutions	50 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	888848848	\$ 6 6 5 4 4 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5
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ا ا	ês	Max	***********		***************************************
٤	precsure (PSI)	Min	98 55 55 55 55 55 55 55 55 55 55 55 55 55	25 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5	5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5
		Max	1625 1525 1640 1575 1600 1600 1575 1625 1625 1625 1625 1625 1625	1525 1525 1525 1526 1530 1530 1530 1530	1550 1550 1550 1550 1550 1550 1550 1550
	3	(seconds)			
		(minutes)	* * * * * * * * * * * * * * * * * * * *	*******	\$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$
		number	164 165 166 167 169 170 171 171 173 174	173 178 179 180 181 182 183	134 188 199 199 200 201 201 201 201 201 201 201 201 201

		Remarks	Pin measured at 0,395 inoh at 230 PSI	Pin measured at 0, 307 inch at 250 PSI	Pin measured at 0.424 inch at 250 PSI	Pin measured at 0,440 inch at 256 PSI	Pin measured at 0,440 inch at 250 PSI			
-		Max	<u> </u>	ã.	<u>ā</u>	ž.	<u> </u>			
	Pressure plate	Start								
	\vdash									
]	Piston housing	Start Max								
tures	+	Max					· · · · · · · · · · · · · · · · · · ·			
Temperatures	Rotor 4	Start								
۴		Max					<u></u>			
	Rotor 3	Start								
	Rotor 2									
	Rotor 1	Start								
	1	cevolutions	# 4 4 4 5 4 4 4 4 4 5 8 8	2 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	5 2 2 2 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3	\$ 4 4 8 5 5 6 4 6 5 6 5 6 5 6 5 6 5 6 5 6 5 6 5	2 20 E	88:	2 2 2 2
	(lbs x 10-4)	Min		22222222	122222222	1222	01 00 00 00 00 00 00 00 00 00 00 00 00 0	သတယ	ၿ၈	ာ တ လ တ
	9 <u>6</u> 9	Max	*****			8888	22222	នគន	888	ន្តន្តន្ត
إ	pressure (PSI)	MB	888888888888888888888888888888888888888		8888888888					8 2 8 8
L		Max	1525 1525 1525 1700 1700 1475 1475 1475 1475 1400 1350 1350 1350 1350	1450 1450 1450 1450 1450 1450 1450 1450	1425 1425 1500 1450 1450 1350 1325 1325	1458 1478 1478	1500 1450 1450 1450	1475	122	1475 1350 1325
		(speconds)					1111111	11:3	11.3	
	Time		÷ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$	t	* * * * * * * * * * * * *	: 1 & & &	វៈ វៈ វៈ វៈ វៈ	\$ \$ \$	\$ \$ \$.
		number	222 222 222 222 222 222 222 222 222 22	2	252 253 253 253 253 253 253 253	8 28 88	262 263 263 264 264 263	266 267	268 269	272

		Remarks	Pin measured at 0,488 inch at 280 PCI	Pin ressured at 0, 470 inch at 250 PSI	Pin measured at 0,481 inch at 250 PH	hatrument air pressure was low	Pin messured at 0,486 inch at 250 PSI	Pin measured at 0,610 inch at 280 PSI
	ressure	Max						
	Pressure plate	Gert						
	Piston housing	Max						
	Pie bou	Start						<u> </u>
Temperatures	Rotor 4	Max						
F.	Ro	Start						
	Rotor 3	t Max						
	Ro	Regre						
	Rotor 2	rt Max						
	E .	x Start						
	Rotor 1	Start Max						
		revolutions	2000 4 4 50 50 50 50 50 50 50 50 50 50 50 50 50	8888888888	9 4 4 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5	2 2 2 3 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	0 4 7 4 4 4 7 4 7 0 0 0 0 0 0 0 0 0	25 25 25 25 25 25 25 25 25 25 25 25 25 2
		Mfn	8 8 9 10 10 10	2222000	თ თ თ თ თ თ თ			ೲೲೲೲೲೲ۲۲ೲೲೲ ೲ
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	pressure (PSI)	Min						888288882888888888888888888888888888888
		Max	1425 1440 1440 1450 1450 1450	1450 1450 1450 1450 1450	1450	128881188813 128881188813	1450 1450 1455 1455 1455 1455 1455	1450 1450 1450 1450 1450 1450 1450 1450
		(seconds)						44664664666
	Time	(minutes)	8 4 8 4 4 4 4	***	\$\$00\$\$\$	t	* * * * * * * * * * * * * * * * * * *	\$ \$ \$ \$ \$ \$ \$ 6 \$ \$ \$ \$ \$ \$ \$ \$
		number	27.7 27.7 27.8 27.8 280 281	282 284 285 287 288	284 282 283 284 284 284 284 284 284 284 284 284 284	200 7 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3	300 300 300 300 300 300 300 300 300 300	316 317 319 320 321 323 324 325 326 326 326 327

		Remarks	Pin measured at 0,553 inch at 250 PSi. Pin measured at 0,553 inch at 250 PSI	Pin measured at 0,080 mon at 250 Por	Pin measured at 0, 617 inch at 250 PSI
	Prossure plate	T Max			
	2 d	Start			
1	Piston housing	Start Max			
rature	Rotor 4	Max			
Temperatures	Rot	Start			
	Rotor 3	Max			
	Rot	Start			
	Rotor 2	Max			
	Ro	Start			
	Rotor 1	Start Max			
	Florentan	revolutions	222222222222222222222222222222222222222	5533	52 51 50
end	(lbs x 10-4)	Min	ಀಀ ಀಀೲಀಀಀಀಁಀಁಀಁಀಁಀಁಀಀಀಀಀಀಀಀಀಀೲೲೲೱೱೱೱೱೱೱೱೱೱ	~~~	8888
Į į	€2	Max			
l sk	pressuro (PSI)	Min	86 99696 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2		
F 6		Max	1450 1450 1450 1450 1450 1550 1550 1550	155.	165 155 162 160
		(seconds)			11.5
	Time	(minutes)	**	2 G 2 4	60 60 45 45
	į	number	23.23.23.23.23.23.23.23.23.23.23.23.23.2	376 377 378 379	380 381 383

		Remarks			Pin measured at 0.630 inch at 250 PSI		Pin measured at 0,654 inch at 250 PSI			Din measured at 0, 658 tuch at 250 PSI						Pin measured at 0,676 inch at 250 PSI Pin measured at 0,678 inch at 250 PSI. Tost terminated because of torque docrease.
	Pressure plate	Start Max														
	Piston housing	Start Max							<u>-</u> .							
Temperatures	Rotor 4	Start Max														
Ŧ	Rotor 3	Start Max		· · · · ·				<u> </u>								
	Rotor 2	Start Max														
	Rotor 1	Start Max			. <u>.</u> "											
	Floorbool	revolutions	552 522 50 50	52220	25022	2222	2 2 2 2	22 22	55 55	53 52 53	2225	523	523	3 2	52.52	77
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ARRESTING SYSTEM						
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13. ABSTRACT						

This report presents the results of a comparison test between a conventional B-52 disk brake and a newly developed sintered metal brake. The conventional B-52 brake, with cerametallic linings, is currently used on the B-52 aircraft and on the BAK-9 and BAK-12 aircraft arresting systems. The newly developed brake has sintered metal brake linings and is designed for the same applications as the conventional brake.

The objective of the test program was to compare the useful life and operating characteristics of the conventional brake with that of the sintered metal brake under conditions of simulated arrestments. The laboratory controlled conditions simulated a BAK-12 arrestment of a 35,000-pound aircraft at a speed of 150 knots.

The conventional brake yielded a useful life of 166 arrestments and the sintered metal brake had a life of 427 arrestments. The coefficients of friction for both brakes remained relatively constant throughout the test program.



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14. KEY WORDS	, LIN	, LINK A		LINK B		LINKC	
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